

# Constraint Equations

# **Lesson 2**

## **Constraint Equations**

# Constraint Equations

Kinematic Constraint notation

$$\Phi^K(\mathbf{q})$$

Driving Constraint or motion in Ansys motion

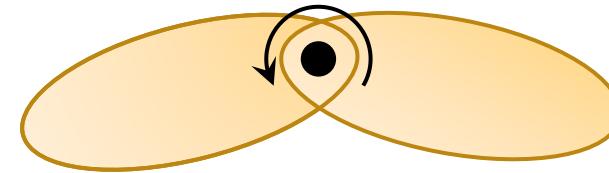
$$\Phi^D(\mathbf{q}, t) = x - \sin(t) = 0$$

Combined Constraint equations

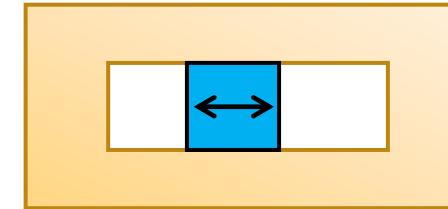
$$\Phi = \begin{bmatrix} \Phi^K(\mathbf{q}) \\ \Phi^D(\mathbf{q}, t) \end{bmatrix} = 0$$

# / Joint Library

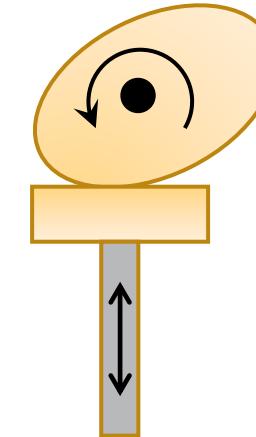
1. Revolute joint



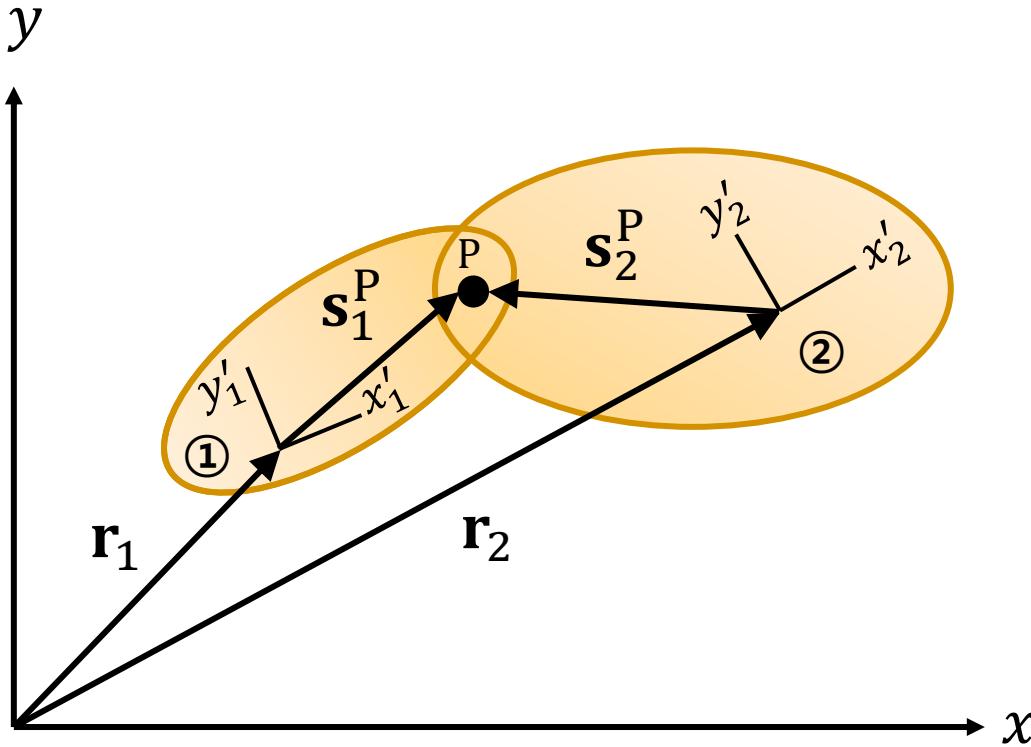
2. Translate joint



3. Cam



# Revolute Joint: Position Level Constraint



$$\begin{aligned}\Phi^{r(1,2)} &= \mathbf{r}_1 + \mathbf{s}_1^P - \mathbf{r}_2 - \mathbf{s}_2^P \\ &= \mathbf{r}_1 + \mathbf{A}_1 \mathbf{s}'_1^P - \mathbf{r}_2 - \mathbf{A}_2 \mathbf{s}'_2^P = \mathbf{0}\end{aligned}$$

# Revolute Joint: Velocity Level Constraint

$$\Phi = \begin{bmatrix} x_1 \\ y_1 \end{bmatrix} + \mathbf{A}(\theta_1) \dot{\mathbf{s}}_1 - \begin{bmatrix} x_2 \\ y_2 \end{bmatrix} - \mathbf{A}(\theta_2) \dot{\mathbf{s}}_2 = \mathbf{0}$$

Differentiating the constraint

$$\begin{aligned}\dot{\Phi} &= \begin{bmatrix} \dot{x}_1 \\ \dot{y}_1 \end{bmatrix} + \mathbf{B}(\theta_1) \dot{\mathbf{s}}'_1 \dot{\theta}_1 - \begin{bmatrix} \dot{x}_2 \\ \dot{y}_2 \end{bmatrix} - \mathbf{B}(\theta_2) \dot{\mathbf{s}}'_2 \dot{\theta} = 0 \\ &= \frac{\begin{bmatrix} 1 & 0 & -\sin \theta_1 \xi_1 - \cos \theta_1 \eta_1 & -1 & 0 & \sin \theta_2 \xi_2 + \cos \theta_2 \eta_2 \\ 0 & 1 & \cos \theta_1 \xi_1 - \sin \theta_1 \eta_1 & 0 & -1 & -\cos \theta_2 \xi_2 + \sin \theta_2 \eta_2 \end{bmatrix}}{\substack{\text{Jacobian Matrix} \rightarrow \mathbf{J}}} \begin{bmatrix} \dot{x}_1 \\ \dot{y}_1 \\ \dot{\theta}_1 \\ \dot{x}_2 \\ \dot{y}_2 \\ \dot{\theta}_2 \end{bmatrix} = \mathbf{0}\end{aligned}$$

$$\frac{\partial \mathbf{A}}{\partial \theta} = \mathbf{B} = \begin{bmatrix} -\sin \theta & -\cos \theta \\ \cos \theta & -\sin \theta \end{bmatrix}$$

# Revolute Joint: Acceleration Level Constraint

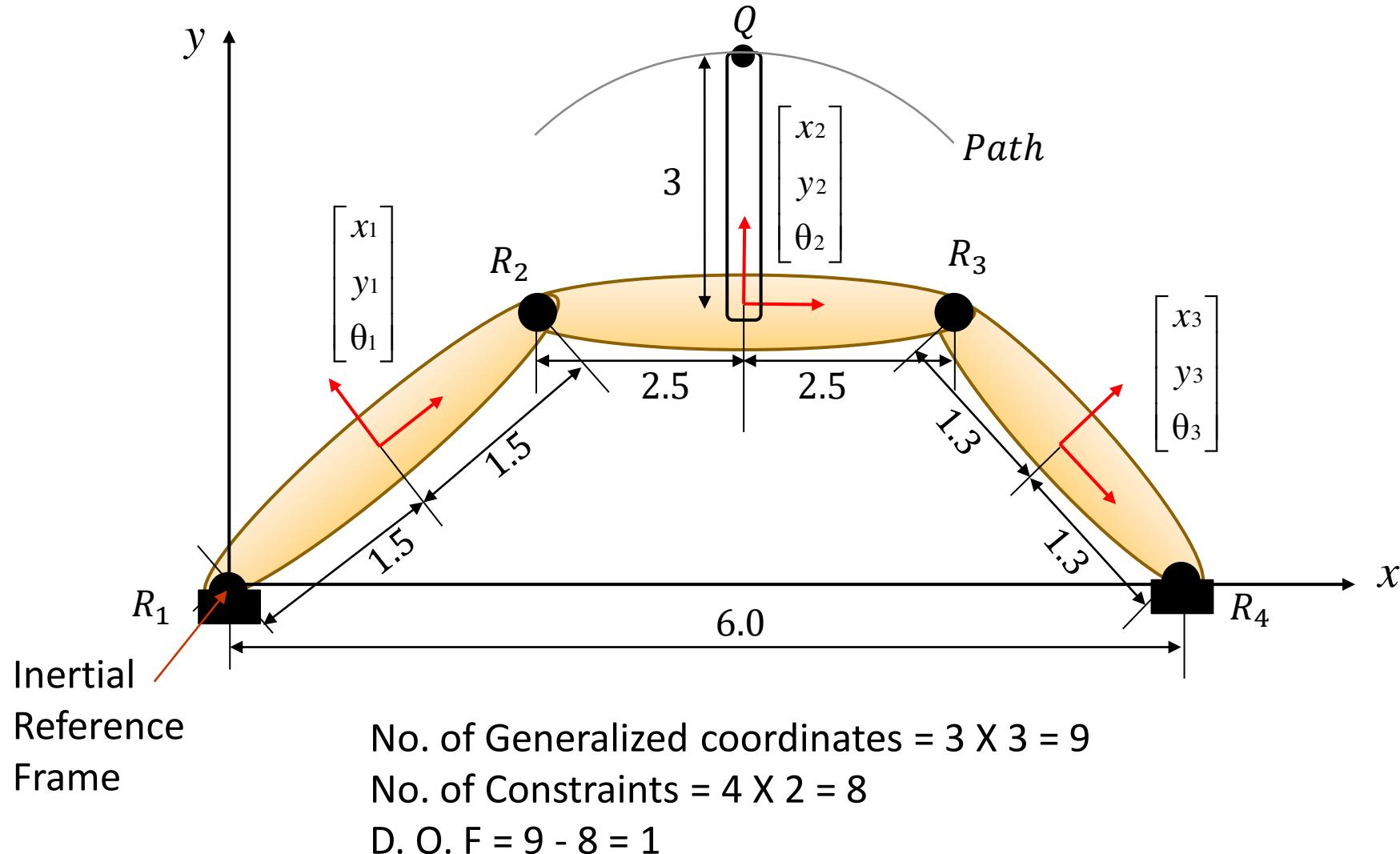
Taking the second time derivative yields

$$\ddot{\Phi} = \mathbf{J}\ddot{\mathbf{q}} + [0 \quad -\mathbf{A}(\theta_1)\mathbf{s}'_1\dot{\theta}_1 \quad 0 \quad \mathbf{A}(\theta_2)\mathbf{s}'_2\dot{\theta}_2] \begin{bmatrix} \dot{x}_1 \\ \dot{y}_1 \\ \dot{\theta}_1 \\ \dot{x}_2 \\ \dot{y}_2 \\ \dot{\theta}_2 \end{bmatrix} = \mathbf{0}$$

$\gamma$  is defined as

$$\mathbf{J}\ddot{\mathbf{q}} = \mathbf{A}(\theta_1)\mathbf{s}'_1\dot{\theta}_1^2 - \mathbf{A}(\theta_2)\mathbf{s}'_2\dot{\theta}_2^2 \equiv \gamma$$

# A Four Bar Mechanism



# Constraints for the Four Bar Mechanism

## Constraints

$$R_1: \Phi_1 = \begin{bmatrix} x_1 \\ y_1 \end{bmatrix} + A(\theta_1) \begin{bmatrix} -1.5 \\ 0 \end{bmatrix} = \mathbf{0}$$

$$R_2: \Phi_2 = \begin{bmatrix} x_1 \\ y_1 \end{bmatrix} + A(\theta_1) \begin{bmatrix} 1.5 \\ 0 \end{bmatrix} - \begin{bmatrix} x_2 \\ y_2 \end{bmatrix} - A(\theta_2) \begin{bmatrix} -2.5 \\ 0 \end{bmatrix} = \mathbf{0}$$

$$R_3: \Phi_3 = \begin{bmatrix} x_2 \\ y_2 \end{bmatrix} + A(\theta_2) \begin{bmatrix} 2.5 \\ 0 \end{bmatrix} - \begin{bmatrix} x_3 \\ y_3 \end{bmatrix} - A(\theta_3) \begin{bmatrix} -1.3 \\ 0 \end{bmatrix} = \mathbf{0}$$

$$R_4: \Phi_4 = \begin{bmatrix} x_3 \\ y_3 \end{bmatrix} + A(\theta_3) \begin{bmatrix} 1.3 \\ 0 \end{bmatrix} - \begin{bmatrix} 6.0 \\ 0 \end{bmatrix} = \mathbf{0}$$

※ Driving Constraint (Motion)

$$\Phi^D = \theta_1 - (t + \frac{\pi}{3}) = 0 \quad , t \text{ time} \quad \longrightarrow \quad \text{Total 9 constraints}$$

The Ansys logo consists of the word "Ansys" in a bold, black, sans-serif font. To the left of the "A", there is a graphic element composed of two slanted bars: a yellow bar above a black bar.

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